

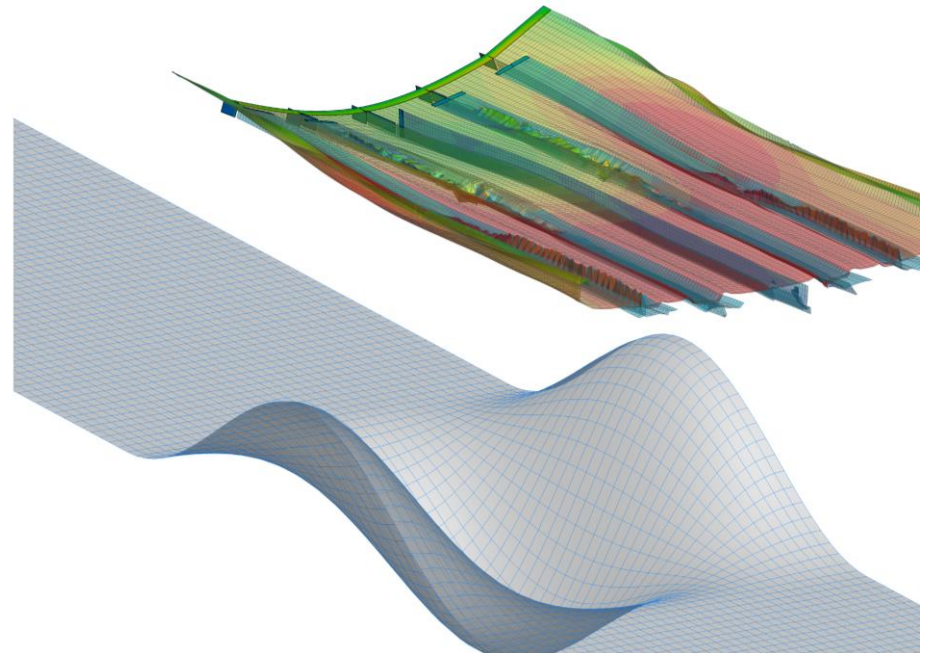
# Deutscher Luft- und Raumfahrtkongress 2019

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## Aeroelastic Analysis of Highly Flexible Aircraft Structures under Large Deformations

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# Motivation

- Large elastic deflections are inherent in particular types of A/Cs: **HALEs, UAVs, Sailplanes**
- Large deformations as the result of aerodynamic optimization
- e.g. Open Class sailplanes with very high aspect ratios
- HALEs and UAVs
  - extreme lightweight and span-loaded design
  - low overall wing deflection at design point but prone to atmospheric disturbances

Aero Vironment Helios, Wing span: 75.3 m



NASA

University of Michigan X-HALE



A2SRL

Concordia Open Class sailplane  
 Wing span: 28 m  
 Aspect ratio: 57  
 $L/D$  max: > 70

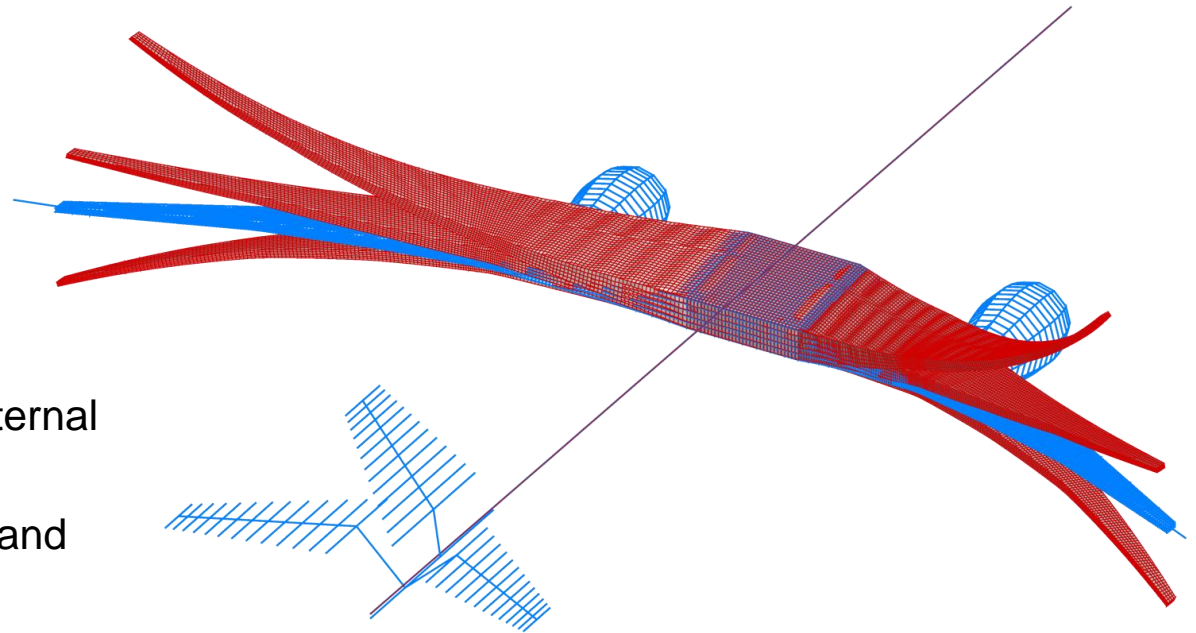


Johannes Dillinger

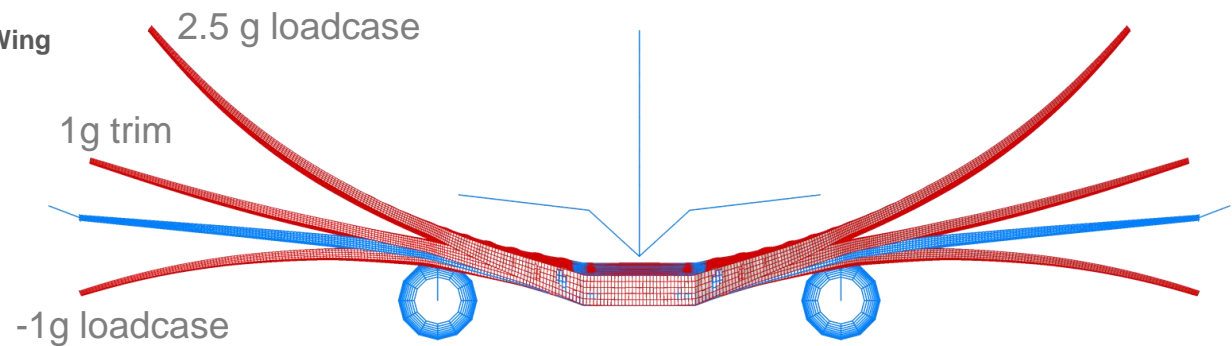
# Motivation

- Jet transport aircraft are entering the regime of large deformations
- Bringing more flexibility into jet transport wings is a future design goal of aircraft industry

- Thinner airfoil  $\rightarrow$  higher  $Ma_{crit}$
- Absorption of energy from external disturbances
- Reduction of structural loads and increase of comfort

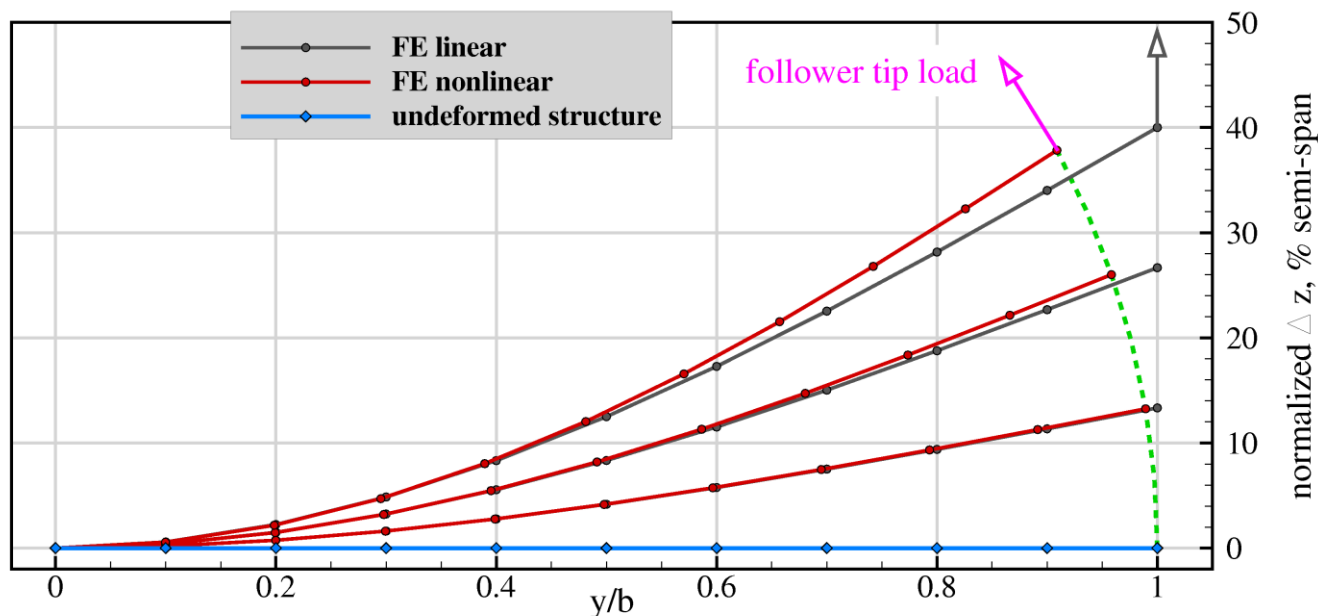


Markus Zimmer, **Design of a Highly Flexible Wing Structure**, ATLAS<sup>2</sup> Hybrid Project



# Problem Statement

- Future aeroelastic analysis and design programs must account for nonlinearities
  - geometrically nonlinear aerodynamics (rotation of forces) ✓
  - geometrically nonlinear structural dynamics (changes in mass and stiffness) ?
  - large rigid-body motions coupled with elastic deflections (stability, disturbances) ?
- Standard analysis and design approaches are not suitable for highly flexible A/C
- Aerodynamic methods are already advanced, but what about structural dynamics?



# State of the Art – Industrial Practice

- Lagrangian methods make the state of the art in nonlinear FE analysis
  - total and updated Lagrangian (TL, UL, e.g. in Nastran *SOL400*)
  - loads are applied stepwise, kinematics and stiffness matrices are updated each step
- High computational costs due to iterative solution
- Commercial FE codes with these methods do not allow for rigid-body motions

## Updating sequence of Lagrangian formulation

$${}^{(n+1)}\mathbf{f} - {}^{(n+1)}\mathbf{r} = 0$$

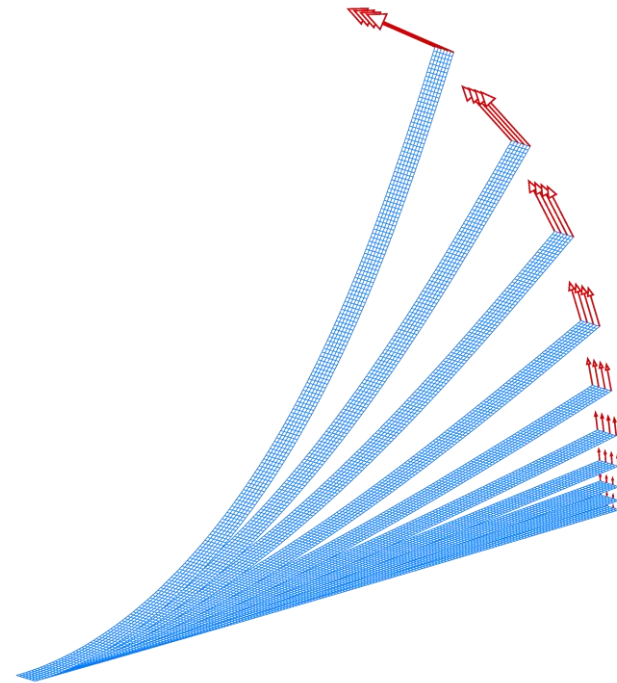
$${}^{(n+1)}\mathbf{r} = {}^{(n)}\mathbf{r} + \Delta\mathbf{r}$$

$$\Delta\mathbf{r} = {}^{(n)}\mathbf{K}^t \Delta\mathbf{u}$$

$${}^{(n)}\mathbf{K}^t = \frac{\partial {}^{(n)}\mathbf{r}}{\partial {}^{(n)}\mathbf{u}}$$

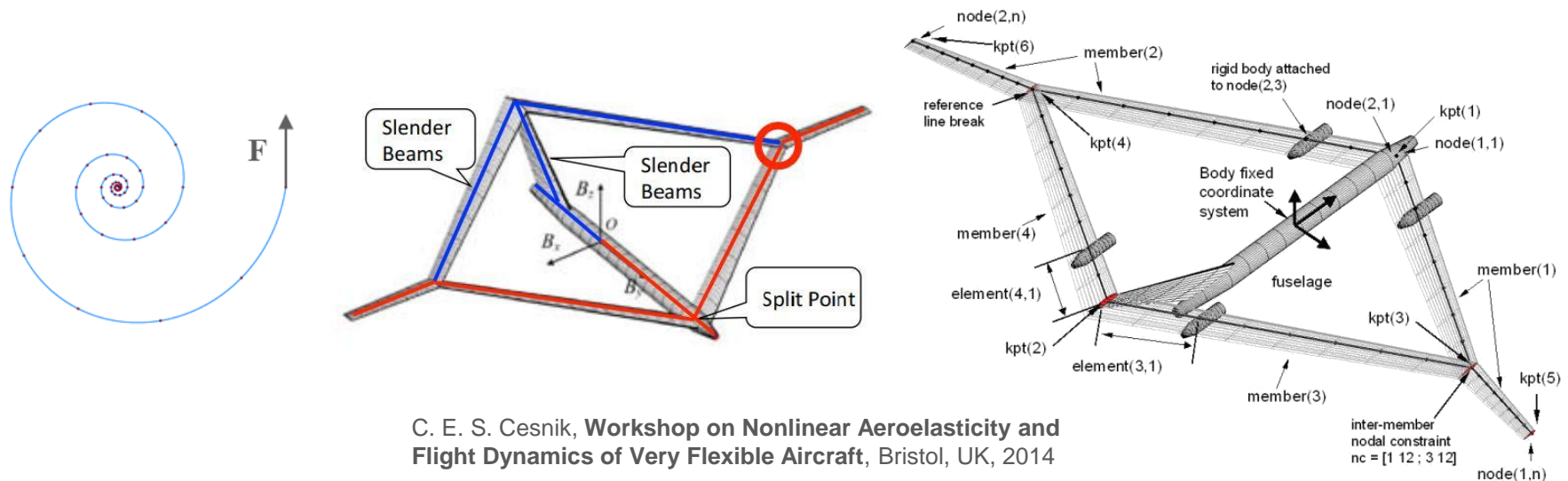
$${}^{(n)}\mathbf{K}^t \Delta\mathbf{u} = {}^{(n+1)}\mathbf{f} - {}^{(n)}\mathbf{r}$$

$${}^{(n+1)}\mathbf{u} = {}^{(n)}\mathbf{u} + \Delta\mathbf{u}$$



# State of the Art – Research

- For flight dynamics of highly flexible free-flying aircraft, mostly *beam* type models are applied
  - *geometrically exact* beam theories (intrinsic, strain-based) are well-established
  - condensation of complex 3D FEM into "equivalent" beam model requires assumptions



- ROMs for nonlinear structural dynamics account for nonlinear load-displacement behavior by *quadratic* and *cubic stiffness terms* and linear and dual modes for displacements<sup>\*</sup>
- Modal expansions in terms of *quadratic modes* are used for rotating and aerospace applications to improve kinematic relations<sup>‡</sup>

<sup>\*</sup> M. P. Mignolet et al., **A review of indirect/non-intrusive reduced order modeling of nonlinear geometric structures**, JSV, 2013

<sup>‡</sup> L. H. van Zyl, E. H. Mathews, **Quadratic Mode Shape Components From Linear Finite Element Analysis**, ASME, 2011



# Research Objective

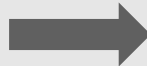
- An improved structural method for geometrically nonlinear aeroelastic analyses is desired
- Main ideas of the new method:

- Consider moderately large deflections,  $\mathcal{O}(< 30\%)$
- Assume nonlinear nodal displacement field is still composed of "modes"
- Identify differences to linear modal approach and find extensions

*Linear modal approach*

$$\mathbf{u} = \Phi \mathbf{q}$$

$$\mathbf{M}\ddot{\mathbf{q}} + \mathbf{D}\dot{\mathbf{q}} + \mathbf{\Omega}^2 \mathbf{q} = \mathbf{Q}$$



*Nonlinear extensions*

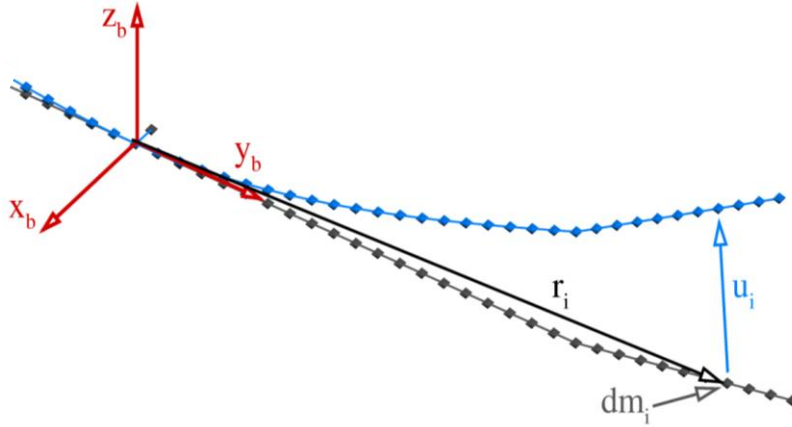
$$\mathbf{u} = \Phi \mathbf{q} + \tilde{\Phi}(\mathbf{q}^2, \mathbf{q}^3, \mathbf{q}^4) \quad (1)$$

$$\mathbf{M}\ddot{\mathbf{q}} + \mathbf{D}\dot{\mathbf{q}} + \mathbf{G}(\mathbf{q}, \mathbf{q}^2, \mathbf{q}^3, \mathbf{f}) = \mathbf{Q} \quad (2)$$

- Subject to:
    - low computational costs (few DOFs: modal space, no iterative solution)
    - applicable to complex/arbitrary FE models
    - easy extension for rigid-body motions
1. Higher-order modal components yield geometrical nonlinearities
  2. Nonlinear stiffness terms for nonlinear force-displacement relations

# Theoretical Derivation: Static Structural Equations

- Geometrically nonlinear displacements are represented by *higher-order mode components*



$$u(q) = \underbrace{{}^p\Phi_0}_{\text{linear}} q_p + \underbrace{{}^p\Phi_1^i}_{\text{quadratic}} q_p q_i + \underbrace{{}^p\Phi_2^{ij}}_{\text{cubic}} q_p q_i q_j + \underbrace{{}^p\Phi_3^{ijk}}_{\text{quartic/fourth order}} q_p q_i q_j q_k$$

mode components: linear quadratic cubic quartic/fourth order

The mode itself becomes a function of the amplitude ( $q$ ):

$${}^p\Phi(q) = {}^p\Phi_0 + 2 {}^p\Phi_1^i q_i + 3 {}^p\Phi_2^{ij} q_i q_j + 4 {}^p\Phi_3^{ijk} q_i q_j q_k$$

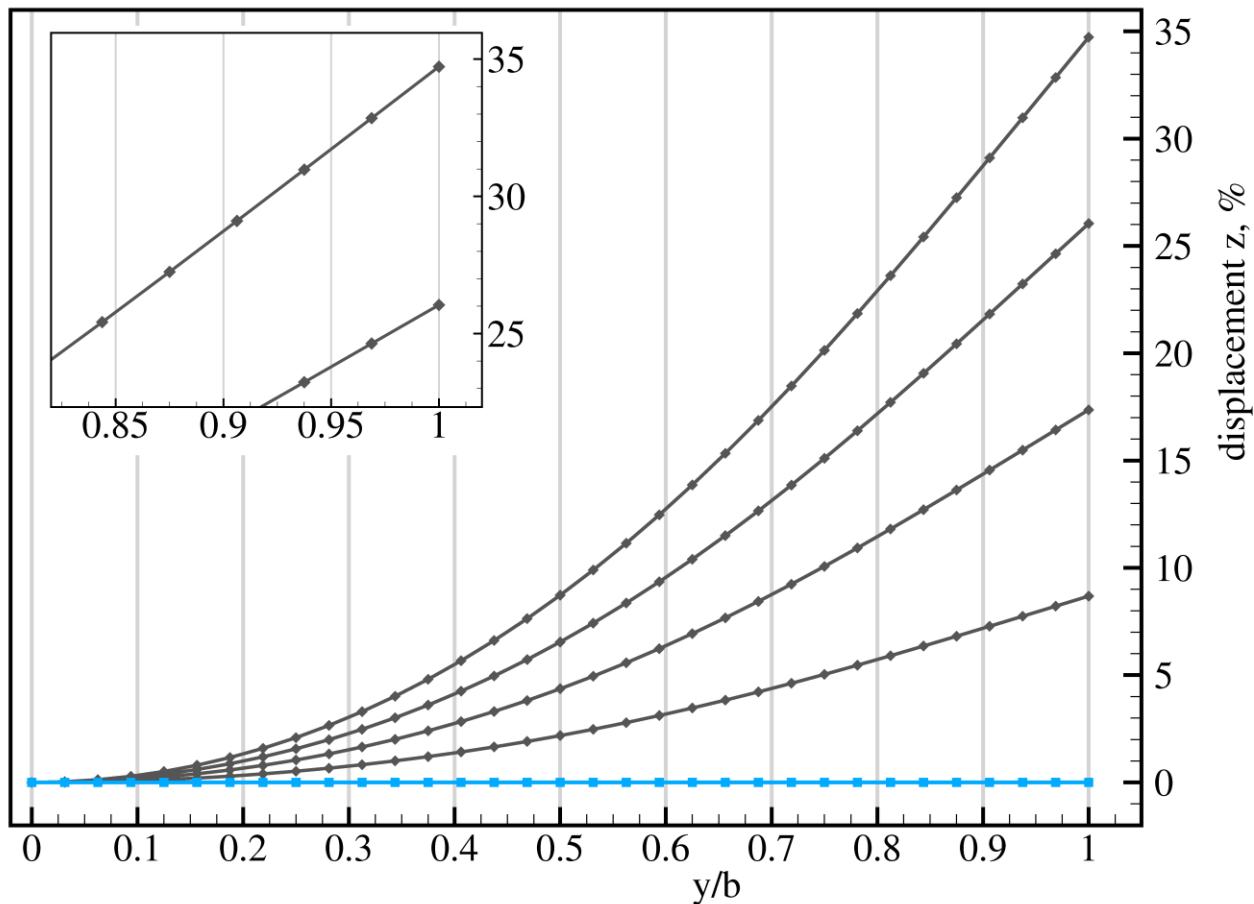
${}^p\Phi_1^i$ ,  ${}^p\Phi_2^{ij}$ ,  ${}^p\Phi_3^{ijk}$  tensors enable a "geometric coupling" of several modes



# Theoretical Derivation: Static Structural Equations

- For illustration: Mode components of the first bending mode of a cantilever beam
- Displacements from linear mode component ("normal mode")

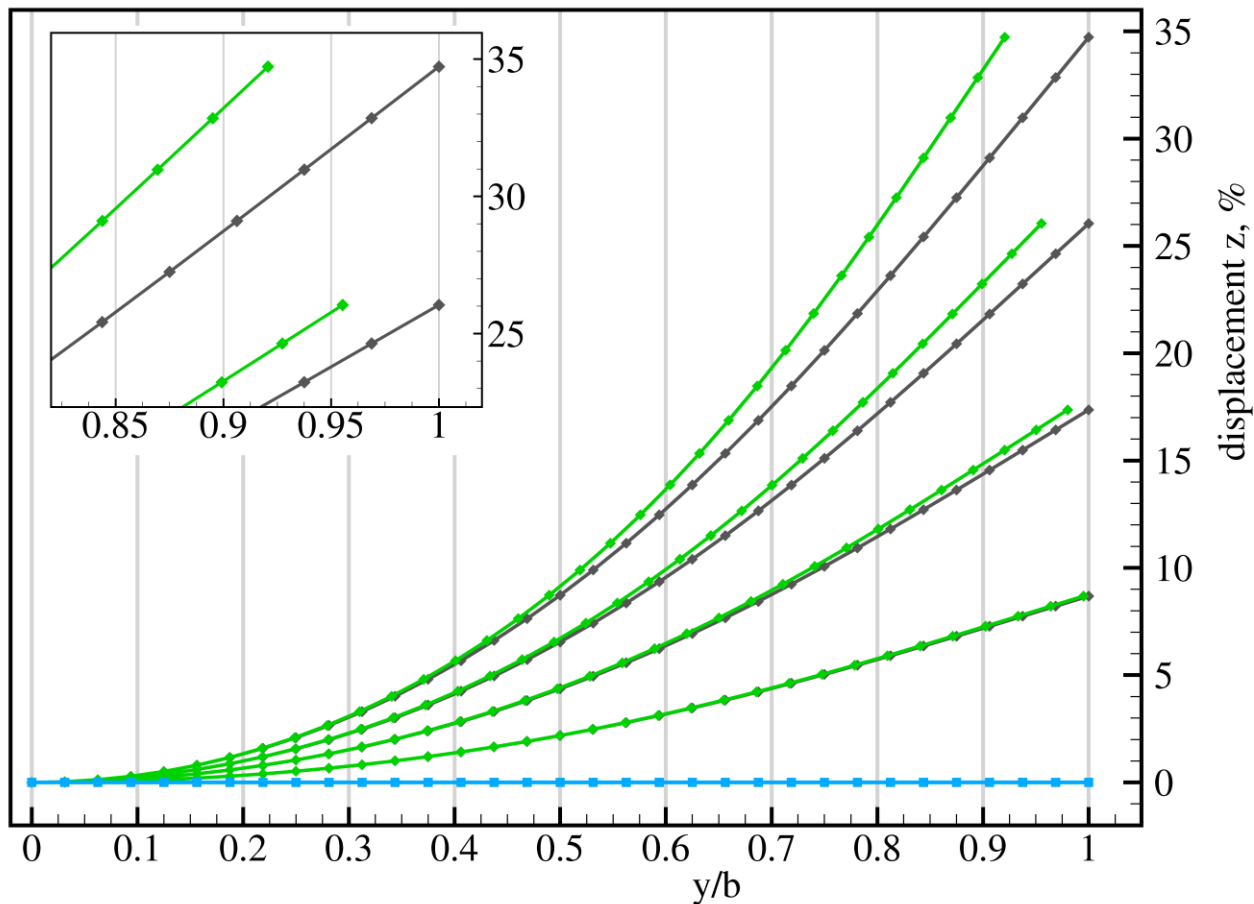
$$\mathbf{u}(\mathbf{q}) = {}^1\Phi_0 q_1$$



# Theoretical Derivation: Static Structural Equations

- For illustration: Mode components of the first bending mode of a cantilever beam
- Displacements from linear and quadratic mode component

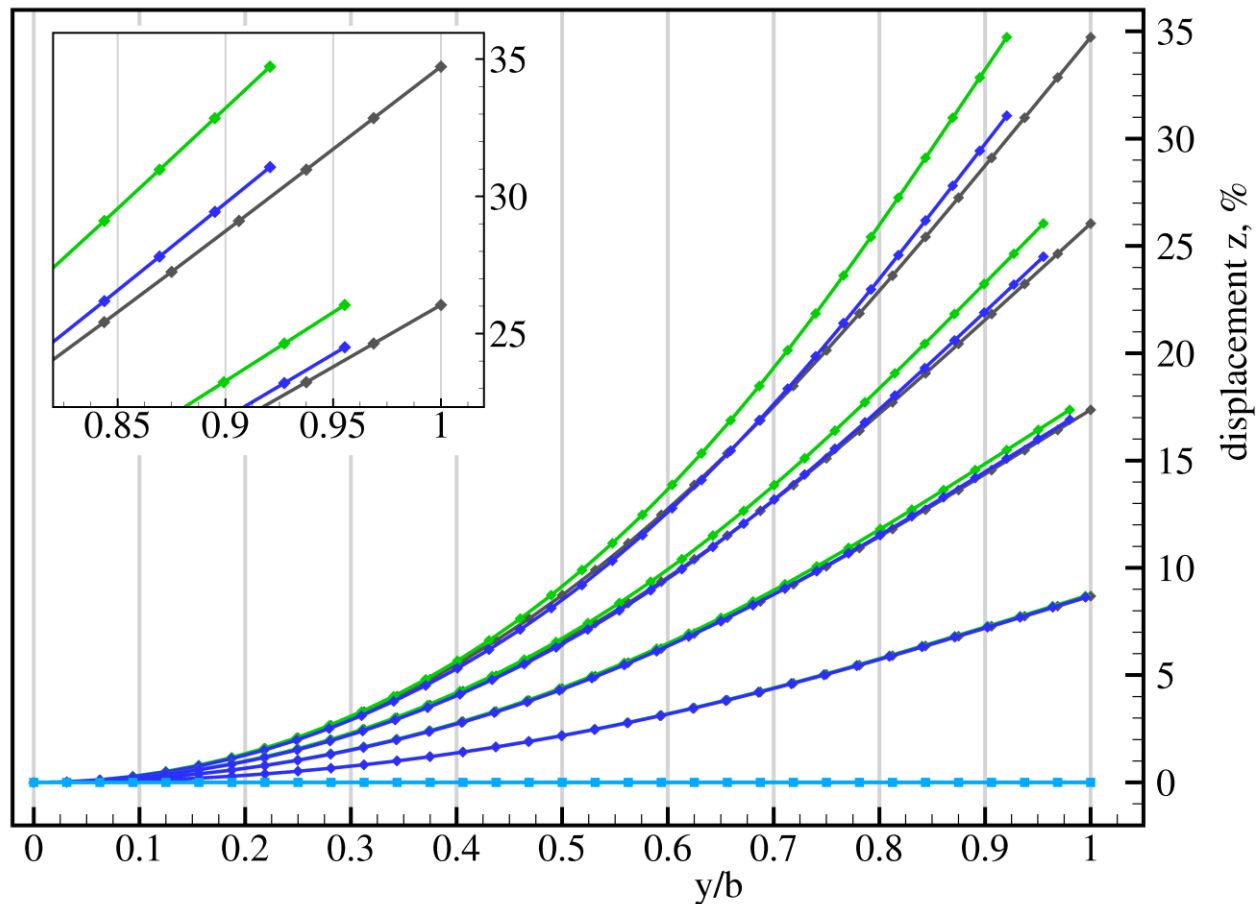
$$\mathbf{u}(\mathbf{q}) = {}^1\Phi_0 q_1 + {}^1\Phi_1^1 q_1 q_1$$



# Theoretical Derivation: Static Structural Equations

- For illustration: Mode components of the first bending mode of a cantilever beam
- Displacements from linear, quadratic, and third order mode component

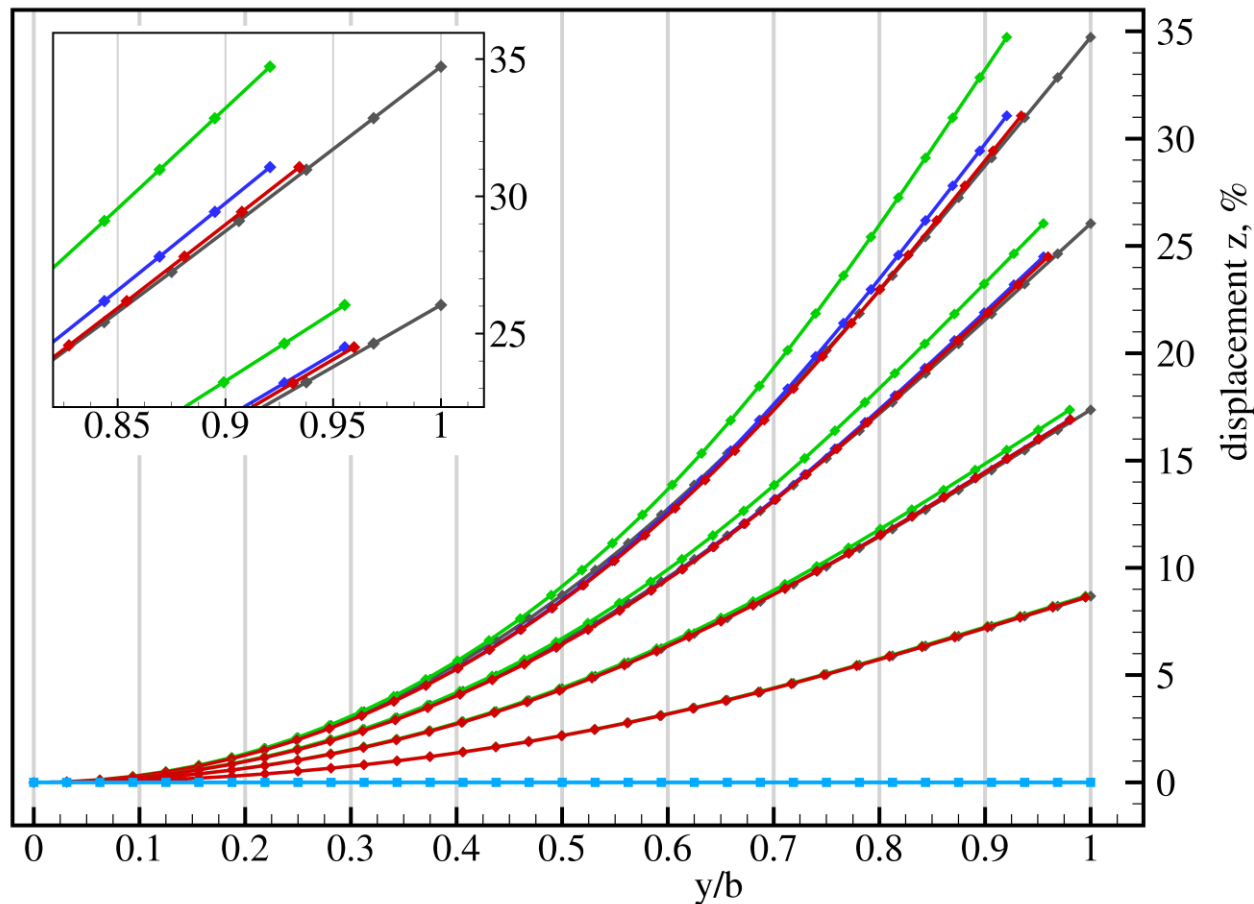
$$\mathbf{u}(\mathbf{q}) = {}^1\Phi_0 q_1 + {}^1\Phi_1^1 q_1 q_1 + {}^1\Phi_2^{11} q_1 q_1 q_1$$



# Theoretical Derivation: Static Structural Equations

- For illustration: Mode components of the first bending mode of a cantilever beam
- Displacements from linear, quadratic, third, and fourth order mode component

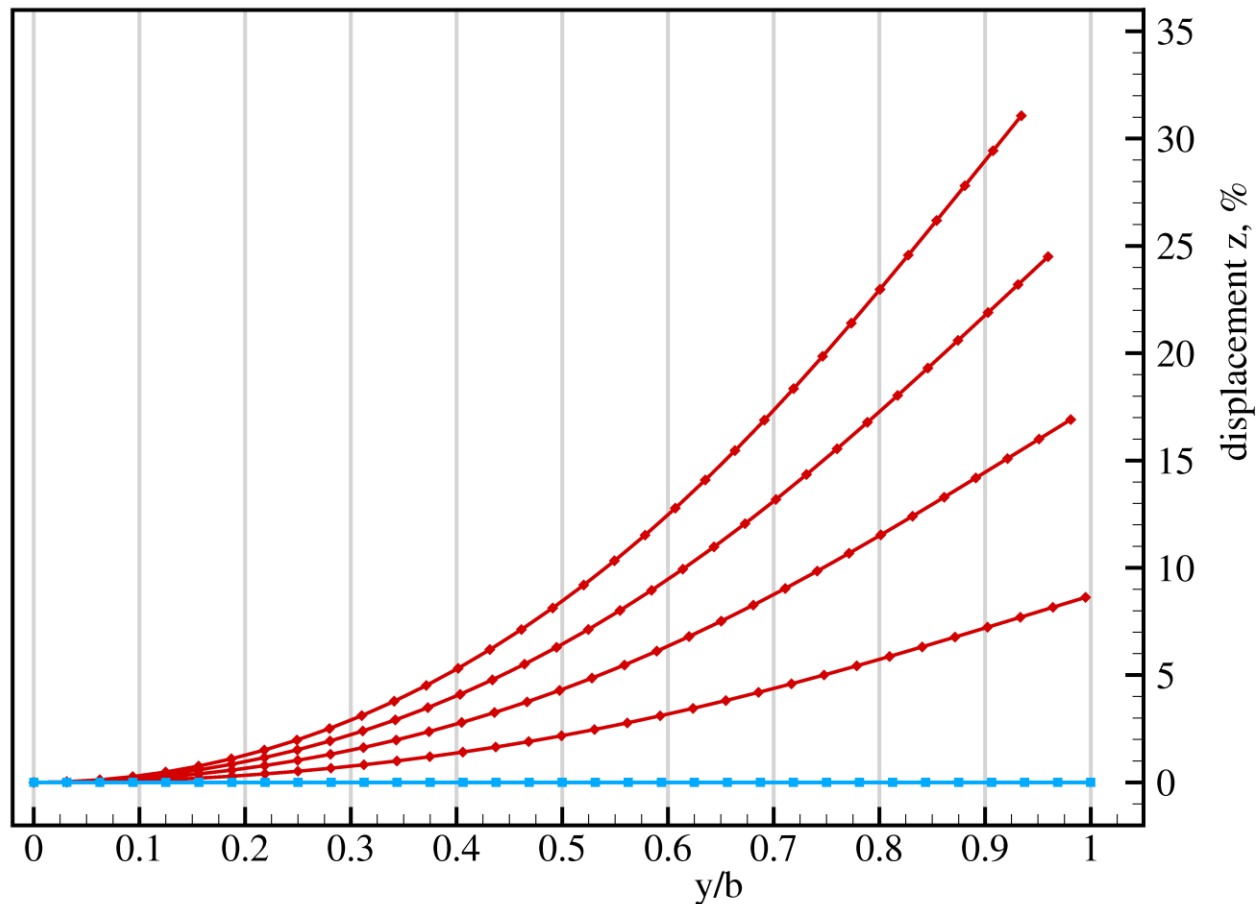
$$\mathbf{u}(\mathbf{q}) = {}^1\Phi_0 q_1 + {}^1\Phi_1^1 q_1 q_1 + {}^1\Phi_2^{11} q_1 q_1 q_1 + {}^p\Phi_3^{111} q_1 q_1 q_1 q_1$$



# Theoretical Derivation: Static Structural Equations

- For illustration: Mode components of the first bending mode of a cantilever beam
- Displacements from linear, quadratic, third, and fourth order mode component

$$\mathbf{u}(\mathbf{q}) = {}^1\Phi_0 q_1 + {}^1\Phi_1^1 q_1 q_1 + {}^1\Phi_2^{11} q_1 q_1 q_1 + {}^p\Phi_3^{111} q_1 q_1 q_1 q_1$$



# Theoretical Derivation: Static Structural Equations

- Expansion of strain energy of a discretized structure in a Taylor series centered at zero (without linear term here):

$$U_s(q) = \frac{1}{2!} \sum_{i=1}^m \sum_{j=1}^m \frac{\partial^2 U_s}{\partial q_i \partial q_j} q_i q_j + \frac{1}{3!} \sum_{i=1}^m \sum_{j=1}^m \sum_{k=1}^m \frac{\partial^3 U_s}{\partial q_i \partial q_j \partial q_k} q_i q_j q_k + \frac{1}{4!} \sum_{i=1}^m \sum_{j=1}^m \sum_{k=1}^m \sum_{l=1}^m \frac{\partial^4 U_s}{\partial q_i \partial q_j \partial q_k \partial q_l} q_i q_j q_k q_l + \text{h.o.t.}$$

$U_s(q)$  = strain energy;  $q_i$  = generalized coordinates

- Application of Castigliano's first theorem yields steady governing equation in generalized coordinates:

$${}^p G_1^i q_i + {}^p G_2^{ij} q_i q_j + {}^p G_3^{ijk} q_i q_j q_k = Q^p ; \quad (p = 1, \dots, m)$$

- Quadratic and cubic stiffness terms account for nonlinear force-displacement relation

${}^p G_1^i$  is comparable to the linear stiffness matrix of the structure (eigenvalues)

${}^p G_2^{ij}$  and  ${}^p G_3^{ijk}$  are tensors of 3<sup>rd</sup> and 4<sup>th</sup> order which enable coupling of several modes

# Theoretical Derivation: Static Structural Equations

- Generalized forces,  $Q^p$ , are calculated by application of the PVW

$$\delta V = \delta \mathbf{u}^T \mathbf{f}$$

- Expanding  $\delta \mathbf{u}$  using the linear and the quadratic mode shape components:

$$\delta V = \delta q_p^T \left( {}^p \boldsymbol{\Phi}_0^T + {}^p \boldsymbol{\Phi}_1^{iT} q_i \right) \mathbf{f}$$

- Generalized forces are then given as:

$$Q^p = {}^p \boldsymbol{\Phi}_0^T \mathbf{f} + {}^p \boldsymbol{\Phi}_1^{iT} \mathbf{f} q_i$$

- Combining yields the final steady governing equation of the proposed method:

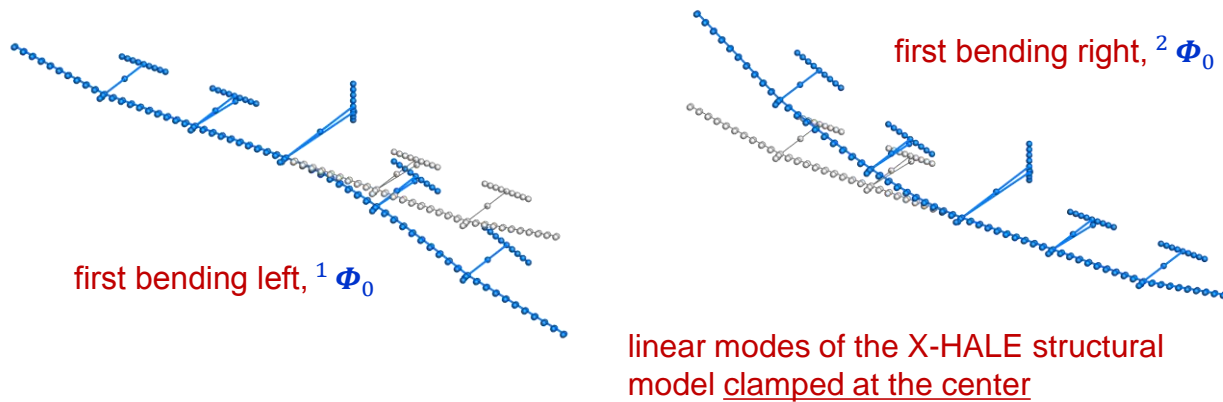
$$\left( \underline{{}^p G_1^i} - {}^p \boldsymbol{\Phi}_1^{iT} \mathbf{f} \right) q_i + {}^p G_2^{ij} q_i q_j + {}^p G_3^{ijk} q_i q_j q_k = {}^p \boldsymbol{\Phi}_0^T \mathbf{f}$$

- Linear stiffness term is a function of the applied loads (comparable to a *tangent stiffness*)



# Theoretical Derivation: EOMs Free-Flying Elastic Aircraft

- The equations derived so far enable geometrically nonlinear structural analysis
- The higher-order stiffness and mode tensors are calculated in pre-processing
- However, they can be calculated for clamped structures only



- Clamped mode shapes require a formulation of flight dynamics EOMs with inertial coupling
- Lagrange's equations of the second kind are used for the derivation
- **No mean axes assumptions**

$$\begin{bmatrix} M_{tt} & M_{tr} & M_{te} \\ M_{rt} & M_{rr} & M_{re} \\ M_{et} & M_{er} & M_{ee} \end{bmatrix} \begin{bmatrix} \dot{V}_b \\ \dot{\omega}_b \\ \ddot{q} \end{bmatrix} + \begin{bmatrix} 0 & D_{tr} & 0 \\ D_{rt} & D_{rr} & D_{re} \\ D_{et} & D_{er} & D_{ee} \end{bmatrix} \begin{bmatrix} V_b \\ \omega_b \\ \dot{q} \end{bmatrix} + \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & \mathbf{G} \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ q \end{bmatrix} = \begin{bmatrix} \tilde{Q}_t \\ \tilde{Q}_r \\ \tilde{Q}_e \end{bmatrix} \quad \begin{array}{l} t = \text{translational} \\ r = \text{rotational} \\ e = \text{elastic} \end{array}$$

$$M, D, G = f(\dot{V}_b, V_b, \dot{\omega}_b, \omega_b, \ddot{q}, q, q)$$

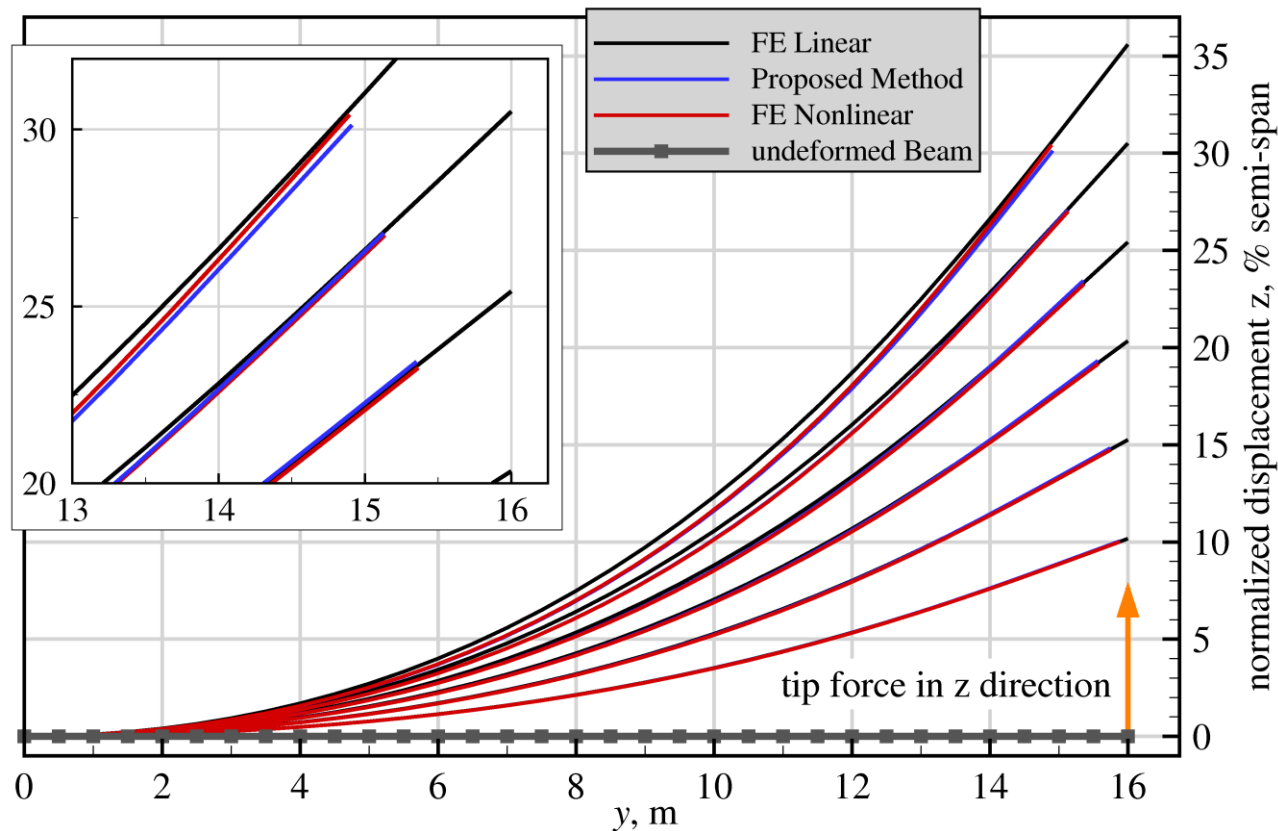
M. Ritter, J. Jones, C. E. S. Cesnik, **Free-flight Nonlinear Aeroelastic Simulations of the X-HALE UAV by an Extended Modal Approach**, IFASD, 2017

# Static Structural Validation: 16m Cantilever Beam

- Static deformation for tip forces along the z direction

$$\left( {}^p G_1^i - {}^p \Phi_1^{iT} f \right) q_i + {}^p G_2^{ij} q_i q_j + {}^p G_3^{ijk} q_i q_j q_k = {}^p \Phi_0^T f$$

$$u(q) = {}^p \Phi_0 q_p + {}^p \Phi_1^i q_p q_i + {}^p \Phi_2^{ij} q_p q_i q_j + {}^p \Phi_3^{ijk} q_p q_i q_j q_k$$



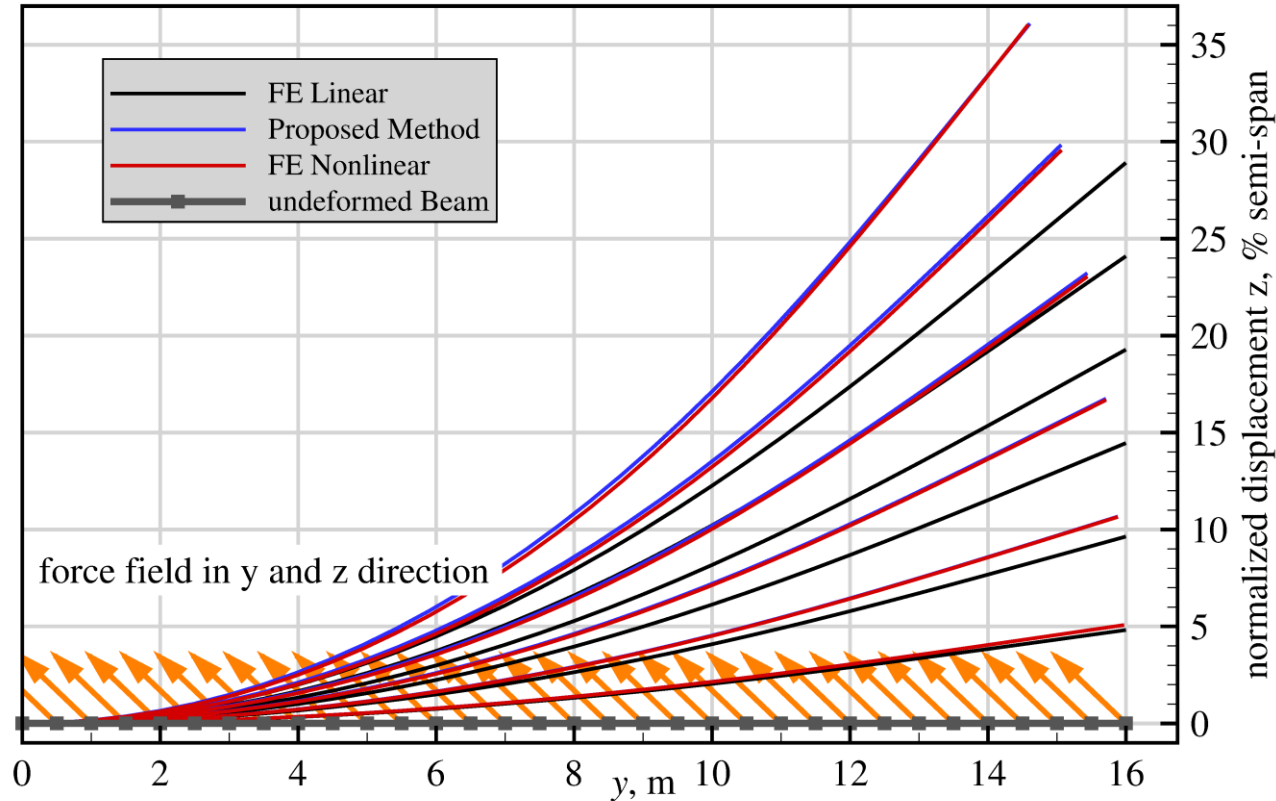
# Static Structural Validation: 16m Cantilever Beam

- Static deformation for tip forces along the z and the y direction

$$\left( {}^p G_1^i - {}^p \Phi_1^{iT} f \right) q_i + {}^p G_2^{ij} q_i q_j + {}^p G_3^{ijk} q_i q_j q_k = {}^p \Phi_0^T f$$

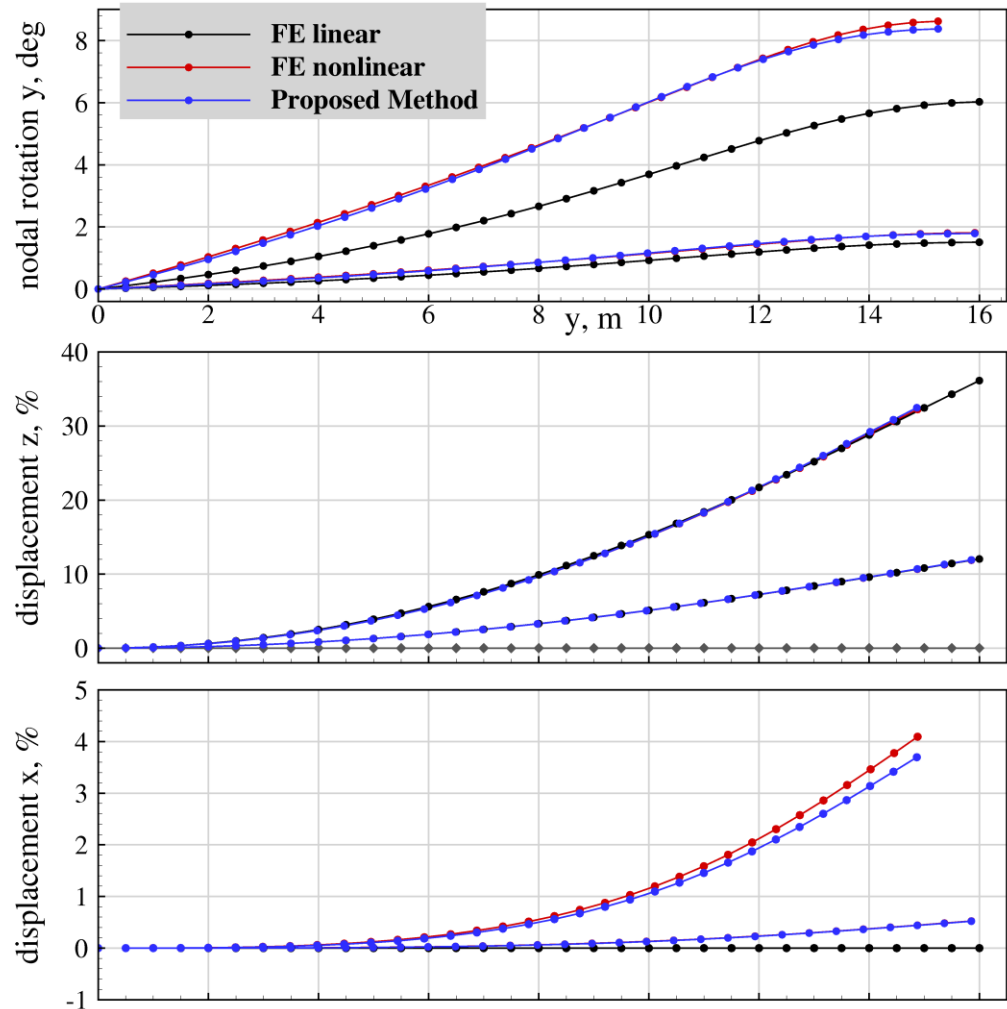
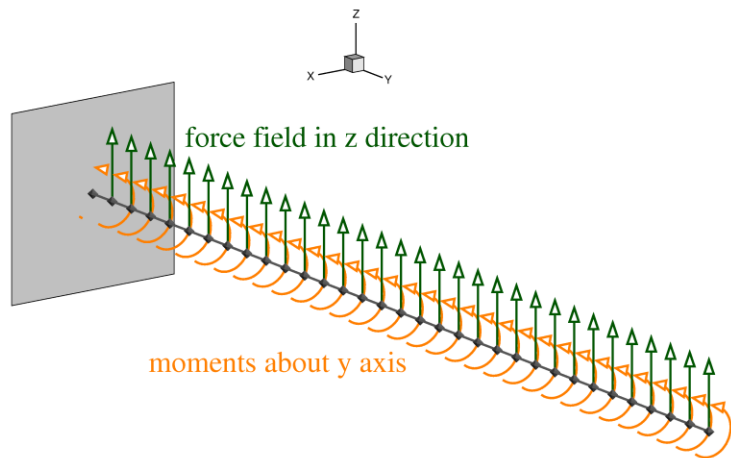
$$u(q) = {}^p \Phi_0 q_p + {}^p \Phi_1^i q_p q_i + {}^p \Phi_2^{ij} q_p q_i q_j + {}^p \Phi_3^{ijk} q_p q_i q_j q_k$$

Linear stiffness term becomes a function of the force field



# Static Structural Validation: 16m Cantilever Beam

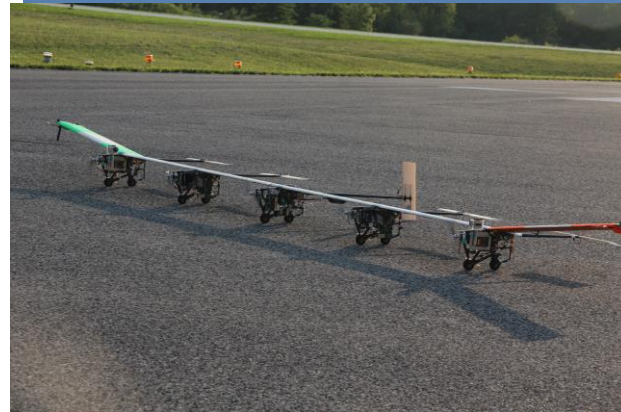
- Static bending deformation for forces along the z direction and moments about the y axis



# Flight Dynamics Simulation: X-HALE UAV

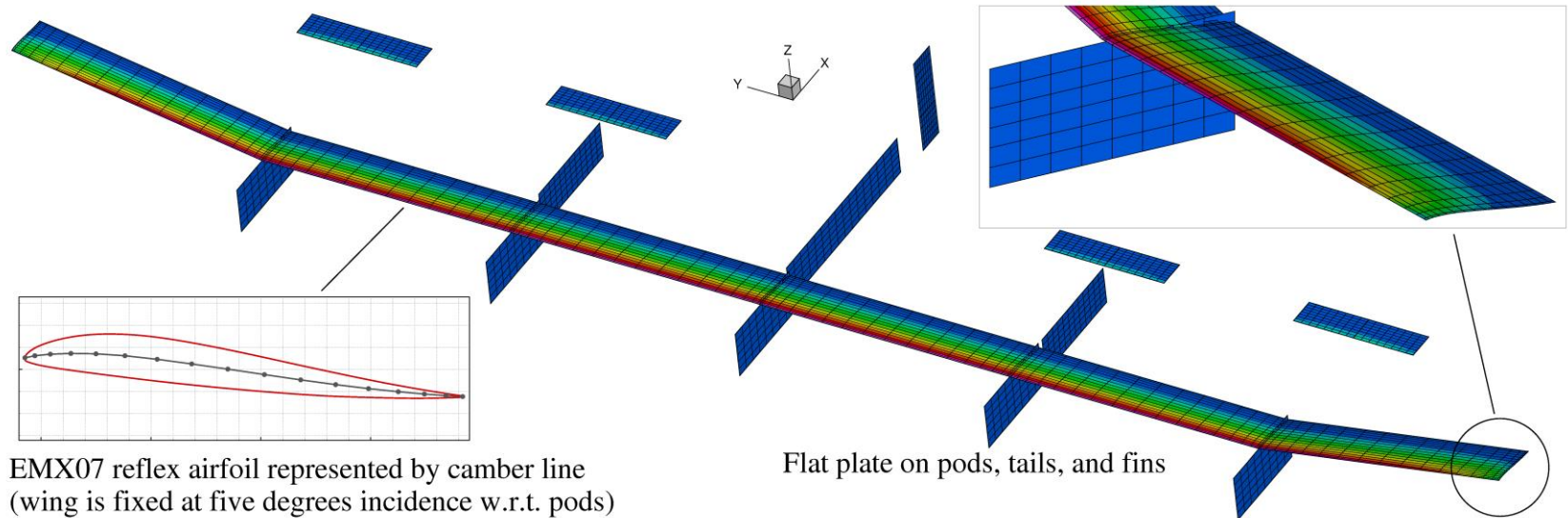
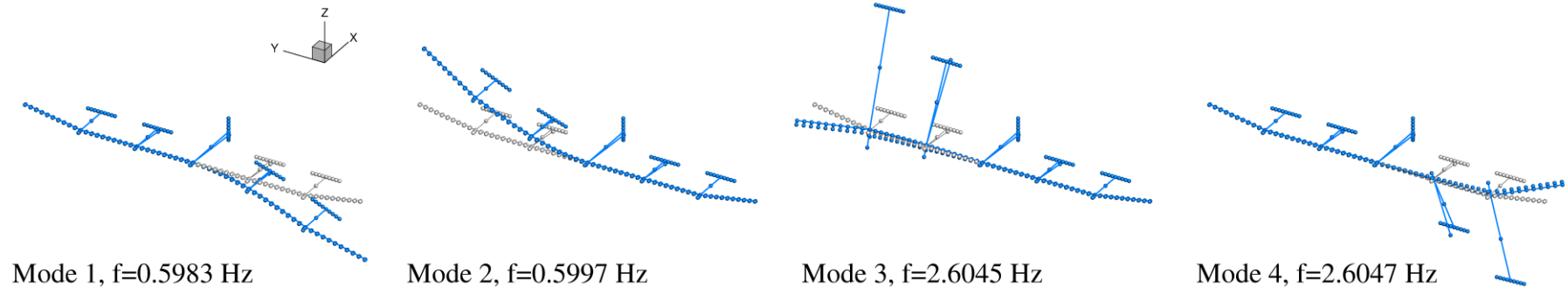
- Properties of the X-HALE UAV from the University of Michigan

Property	<i>X-HALE 6m RRV</i>
Span (wing)	6 m
Mean chord (wing)	0.2 m
Area (wing)	1.2 m <sup>2</sup>
Airfoil (wing)	EMX07 reflex
Aspect ratio (wing)	30
Flight velocity (trim)	16 m/s
Mass	10.86 kg
max. Engine thrust	$5 \times 8.5$ N
Materials	Fiberglass/graphite/foam



# Flight Dynamics Simulation: X-HALE UAV

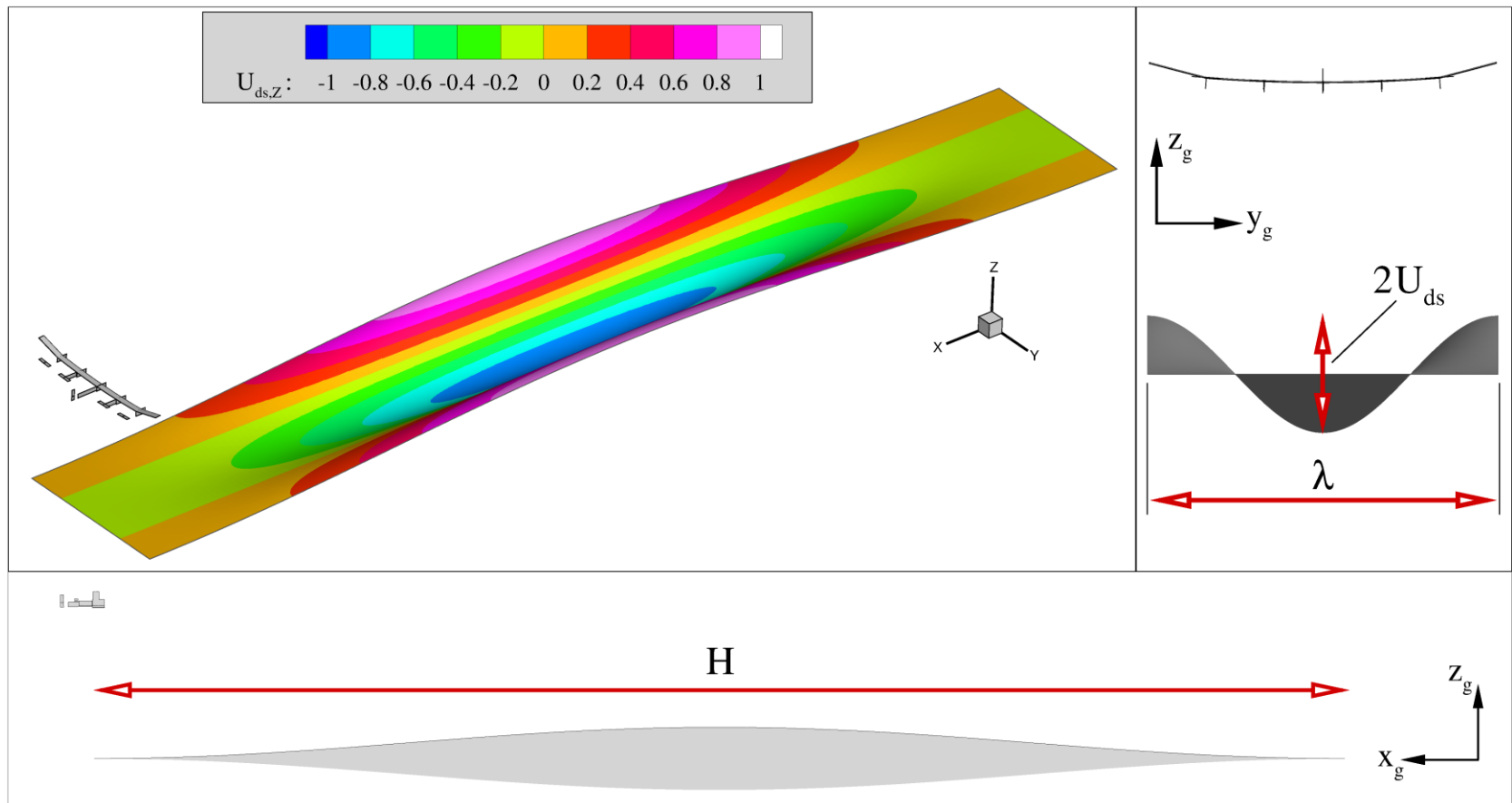
- Nastran FEM with beam elements and discrete masses based on UM/NAST input files
- Mode components calculated in pre-processing appear in pairs left/right



# Application: X-HALE DARPA Gust

- DARPA gust maneuver with  $V_0 = 16$  m/s,  $U_{ds} = -1.5$  m/s,  $H = 40$  m,  $\lambda = 6$  m,  $\Phi = 0$

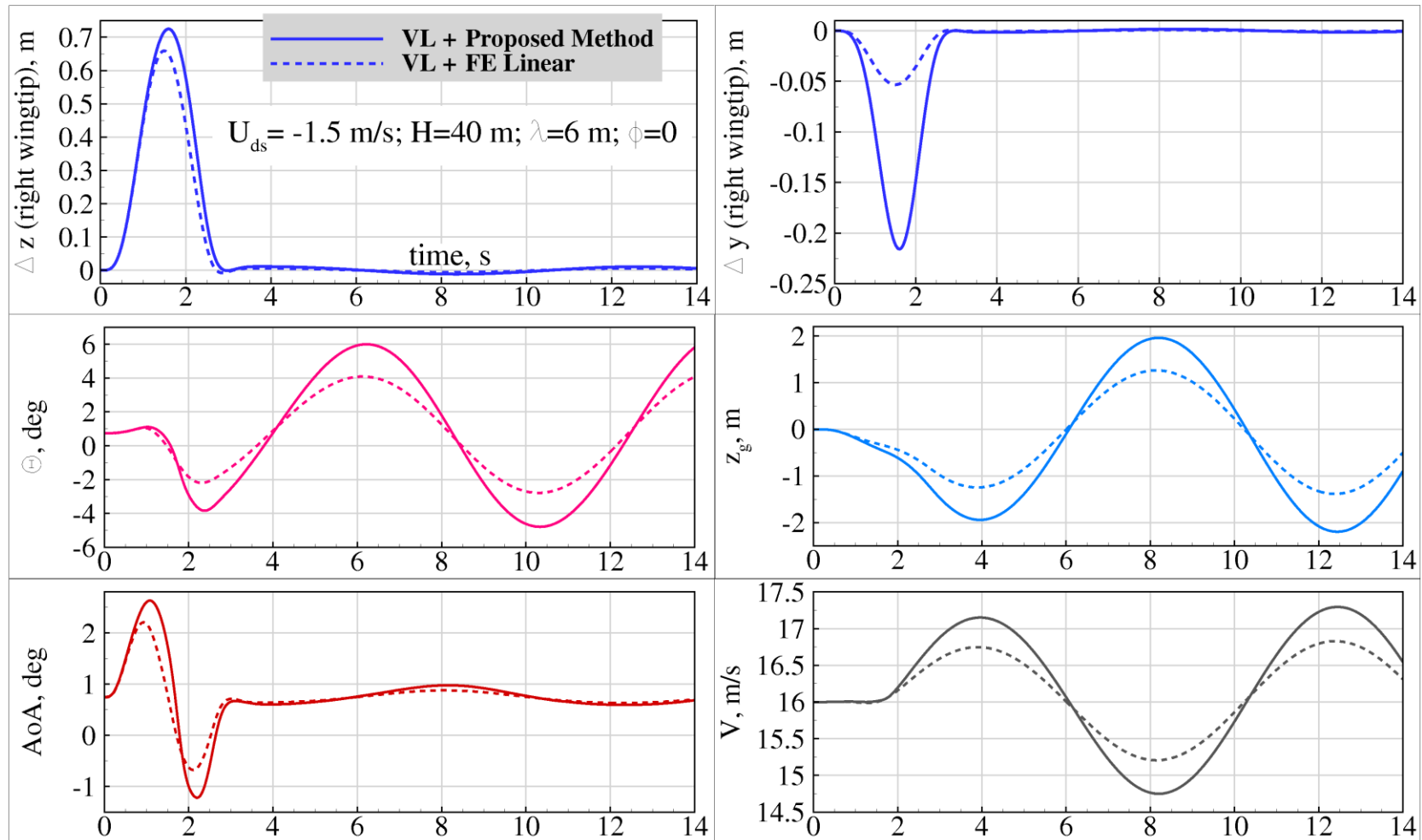
$$U_{ds,z}(x_g, y_g) = \frac{1}{2} U_{ds} \left( 1 - \cos \left( \frac{2\pi x_g}{H} \right) \right) \cos \left( \frac{\pi y_g}{\lambda} - \phi \right)$$



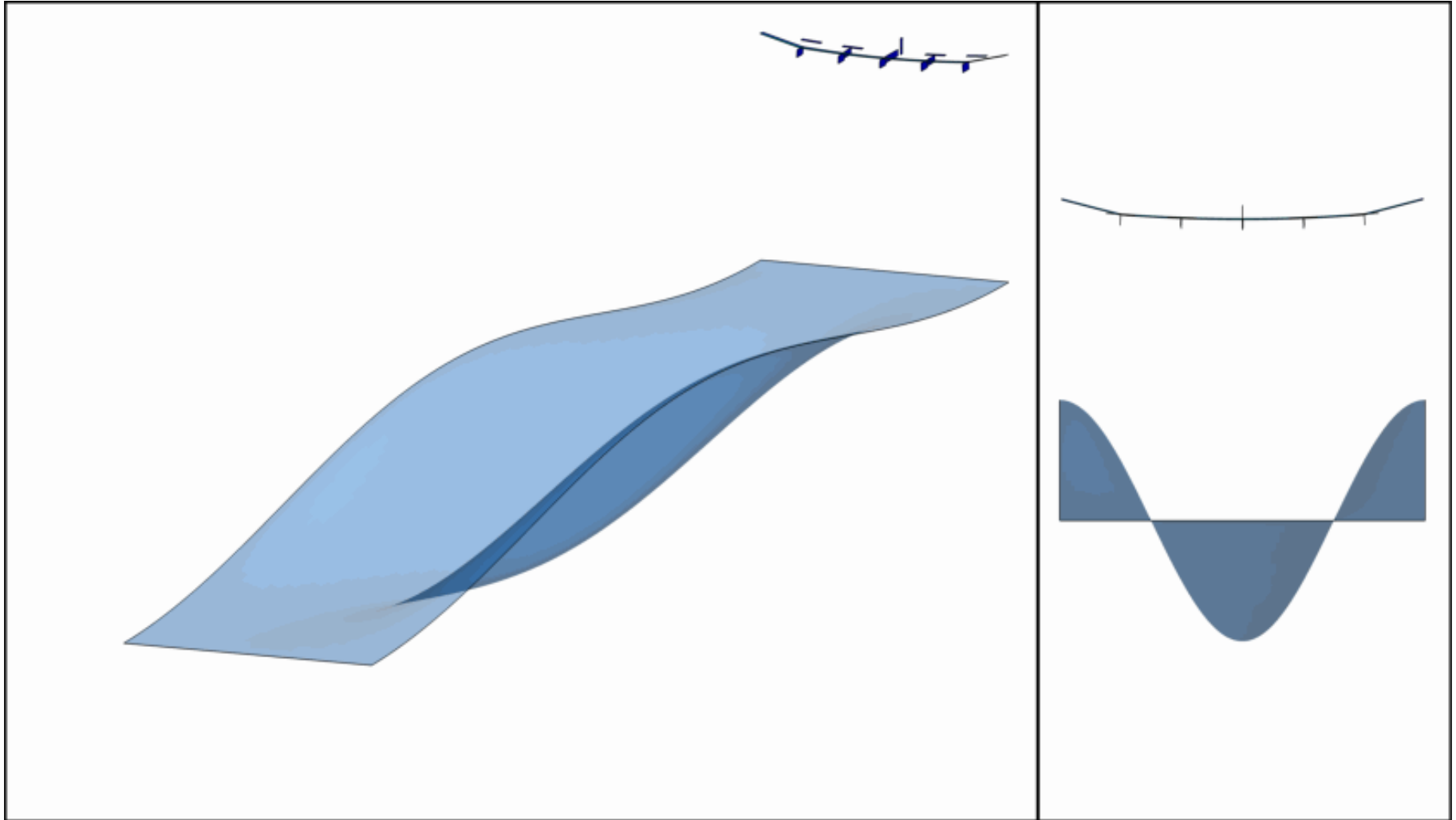


# Application: X-HALE DARPA Gust

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# Application: X-HALE DARPA Gust



# Summary and Outlook

- Development of an improved, modal based method which considers:
  - Geometrical nonlinearities in the nodal displacement field
  - Nonlinearities in load-displacement relation
- Validation with static and flight dynamic test cases
  - Good agreement with static nonlinear reference solutions up to  $\approx 30$  % displacement
  - Good agreement with reference solutions for X-HALE tail input maneuver
- Derivation and implementation of methods for loads recovery
- Modeling and validation of more complex 3D GFEM structures (e.g. jet transport)
- Application of the method to rotating structures such as wind turbines